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A MOLECULAR DYNAMICS SIMULATION OF NANOSCALE WATER VAPOR ABSORPTION ON THE SURFACE OF LIBR AQUEOUS SOLUTION

Hong Hu^{a,*}, J. N. Chung^{a,†}

^a Department of Mechanical and Aerospace Engineering, University of Florida, Gainesville, FL 32611

ABSTRACT

A new architecture for absorption refrigeration systems (ARSs) that enables a significant enhancement of heat and mass transport processes has been proposed. This enhancement in performance is expected to translate into a significant reduction is size and cost of ARSs. The key innovation in the new approach is the use of ultrathin liquid films constrained by highly permeable nanostructured membranes. This approach enables far greater performance than those in the existing macroscale. For example, in the new absorber design, the thin film of LiBr solution is constrained by hydrophobic porous membranes and the inner wall of cooling water channel. The LiBr solution is held outside of the membrane by a meniscus formed over the membrane pores. The solution is cooled by water running in the cooling channel. This paper reports a detailed analysis based on molecular dynamics simulations to obtain a fundamental understanding on the nanoscale transport processes. The effects of LiBr concentration and the water vapor temperature elevation on the condensation coefficient and condensation mass fluxes have been evaluated. A model is also developed to predict the average condensation coefficient for a complete vapor energy spectrum. A comparison between the nanoscale and macroscale condensation mass fluxes is also provided.

Keywords: Molecular Dynamics, Nanoscale, Absorption Refrigeration System

1. INTRODUCTION

Absorption refrigeration systems have been extensively used in commercial facilities due to its ability to directly use low quality waste heat for cooling purposes. However, compared with vapor compression chillers, the volume per unit cooling capacity and the initial installation cost for the absorption refrigeration system are much higher which makes it uncompetitive in small scale applications, even with a low running cost (ASHRAE Handbook, 2010).

Conventional lithium bromide (LIBr) absorption chillers are made of four main components: absorber, generator, evaporator, and condenser. In the absorber or generator, the LiBr solution is sprayed over the surfaces of the columns of pipes inside which cooling water or heat source stream is running. Water vapor is absorbed into the LiBr solution in the absorber and boiled off out of the solution in the generator. The absorption or desorption rate depends on many parameters. But one of controlling factors, as shown in the Nusselt number $Nu = \frac{hD}{k}$ (h is the heat transfer coefficient and D is hydraulic diameter, K is thermal conductivity) is the hydraulic diameter that is also the heat and mass transport length scale. It is clear that one needs to reduce those length scales in the absorber and generator in order to increase the absorption and desorption rates. The use of thin liquid films has shown to be a great potential to

enhance heat and mass transfer in the absorber and generator (Ludwig et al., 2008).

In recent years, membrane based mass exchange devices have been extensively used in separation processes (Lawson and Lloyd, 1997). In Development of a new generation compact and inexpensive solar and waste heat powered absorption refrigeration systems (ARSs) with a substantially better performance than the existing technology will greatly reduce the world energy consumption and carbon emission. Moreover, the ARS fluids considered have no global warming potential (GWP). A new concept that employs the nano-scale architecture for ARSs has the potential to enable an enhancement of the heat and mass transport processes involved within the system by an order of magnitude. This enhancement in performance translates to a proportional reduction in size, weight, and cost of ARSs and their simplification.

The key innovation in the new approach is the use of ultra-thin liquid films constrained by highly permeable nanostructured membranes and surface structures. Ultra-thin films (UTFs) enhance the heat and mass transfer processes (Moghaddam and Ohadi, 2005) in all four system main heat exchangers as mentioned above that account for most of the system volume and weight. This approach enables far greater performance than that achieved in the existing systems where hydrodynamics of falling films over tubes dictates the formation of thick liquid films that impede transport. In the new system architecture, evaporating and condensing refrigerant films are separated from the absorbent UTF by only a thin

the membrane distillation device, the liquid and vapor are separated by a porous membrane where the membrane is not wet by the liquid and no capillary condensation should take place inside the pores of the membrane. At least one side of the membrane should be in direct contact with the process liquid (Lawson and Lloyd, 1997). It has been shown that the membrane distillation has great advantages over traditional distillation methods. The membrane distillation device can be operated with a low energy consumption rate and a minimum expenditure of capital cost and land use (Lawson and Lloyd, 1997).

^{*} Currently with Philips Medical System, Latham, NY

[†] Corresponding author. Email: jnchung@ufl.edu

vapor space and highly permeable nanostructured membranes. In a combined absorber/evaporator, flat panel shape, module (AEM), the absorbent UTF is mechanically constrained by a nanostructured membrane over a surface cooled by the system cooling liquid. The membrane surface facing the absorbent solution is hydrophobic with a pore size range that prevents the solution from entering the membrane. This ensures that the refrigerant molecules encounter minimum diffusion resistance as they pass through the membrane and enter the UTF absorbent. The vapor entering the membrane is generated on surface microstructures, facing the membrane, as the liquid refrigerant absorbs the chilled water heat. The generator/condenser module (GCM) construction is similar to that of AEM except that in this case refrigerant is desorbed from the solution UTF as it is heated by the heating stream. The desorbed refrigerant passes through the membrane and condenses over an enhanced surface cooled by the system coolant.

The new design of the absorber, taking advantages of both the membrane and the ultra-thin film on enhancing the heat and mass transfer, has the typical structure as shown in Fig. 1.



Fig. 1 Schematic of a UTF membrane absorber system.

In the new absorber design, the thin film of LiBr solution is constrained by hydrophobic porous membranes and the inner wall of cooling water channel. The LiBr solution is held outside of the membrane by a meniscus formed over the pores of the membrane. The solution is cooled by water running in the cooling channel. The heights of solution and cooling water channels are very small: on the scales of less than 100µm. The membrane based absorber has been proposed and studied by Ali (2010). In that paper, the results of experimental work are compared with a 1-D model results. The correlations for mass and heat transfer used in that paper are from Martinez and Maroto (2007). The correlations were derived for the mass transfer inside a shell where hollow fibers are woven and uneven flows were suspected. Clearly, the flow patterns are much different between the membrane based absorber and the shell distiller. In the membrane based absorber, the flow is very similar to the Poiseuille flow with very small disturbance in the cross flow direction. While in the distillation module (Martinez and Maroto, 2007), the flow is much complicated in all velocity components and much larger velocity disturbances are from the fibers. Thus, it is not clear whether the correlations in Martinez and Maroto (2007) can be applied in the membrane based absorber.

In this paper, we focus on the physical understanding and the evaluation of the condensation efficiency for the nanoscale absorption of water vapor at the vapor-liquid interface through a condensation process. The paper is organized into two parts: 1. Evaluation of the nanoscale effects of LiBr concentration on the water vapor condensation rate, and 2. Comparison of nanoscale condensation rates with those of macroscale.

2. MD SIMULATION MODEL FOR NANOSCALE VAPOR ABSORPTION ON THE LIBR SURFACE

2.1 Nanoscale Simulation

As mentioned above, the main objective of the current study is to evaluate the effects of solute concentration on the vapor absorption (condensation) in a nanoscale framework using a molecular dynamics (MD) simulation method. Since the first principle approach of molecular dynamics simulation has been chosen as the method of approach, the scale of the model is therefore limited to the nanoscale due to the capabilities of the current computational capabilities. Since the focus of the current study is to investigate the absorption (condensation) of water vapor molecules by the LiBr aqueous solution at the vapor-liquid interface from a nanoscale point of view using molecular dynamics (MD) simulations, the initial configuration of the nanoscale model is shown in Fig. 2. The simulation domain is enclosed in a rectangular box with xyz dimensions of $100\dot{A} \times 25\dot{A} \times 25\dot{A}$ (\dot{A} is angstorm, 10^{-10} m) as plotted in Fig. 2. Initially, the liquid solution of water molecules and ions are placed at random in the middle of the simulation box so as to form a aqueous slab (blue box) with dimensions Of $25\dot{A} \times 25\dot{A} \times 25\dot{A}$ The liquid region is sandwiched by vapor regions and each vapor regions has dimensions of $40\dot{A} \times 25\dot{A} \times 25\dot{A}$



Fig. 2 Schematic of simulation box for the liquid-vapor interface at equilibrium.

2.2 Method of MD Simulation

For any classical molecular dynamics simulation, the force field potential must be determined first. Since the system includes water molecules, Li+, and Br-, the long-range Coulombic force between ions must be added to the Lennard-Jones potential, which can be expressed below with no external potential is applied,

$$U_{ij} = \frac{q_i q_j}{r_{ij}} + 4\varepsilon_{ij} \left[\left(\frac{\sigma_{ij}}{r_{ij}} \right)^{12} - \left(\frac{\sigma_{ij}}{r_{ij}} \right)^6 \right]$$
(1)

Where the subscripts i and j denote ith and jth particles (atoms or ions), q is the charge of particle, r is the distance between particles, σ_{ij} and ε_{ij} are the minimum energy and the zero energy separation distances. The SPC/E water model (Chatterjee et al., 2008) shown in Fig. 3 was used to characterize the water molecules. The mixing rule used in the current paper is called the Lorentz-Berthelot rule given in details in the book by Hanson and McDonald (1976). According to Bouazizi and Nasr (2007) the Lorentz and Berthelot rule has been widely used and worked well with the Lennard-Jones force potential adopted in the current paper.

Lorentz rule was proposed by H.A. Lorentz: $\sigma_{ij} = 0.5(\sigma_{jj} + \sigma_{ii})$. The Berthelot rule is given by $\varepsilon_{ij} = \sqrt{\varepsilon_{ii}\varepsilon_{jj}}$ as follows :

$$\sigma_{ij} = \frac{\sigma_i + \sigma_j}{2} \tag{2}$$



Fig. 3 Schematic of SPC/E water model.

Specifically, it is important to note that for Li+ - Li+ interaction, we adopted $\varepsilon_{Li} = 0.1649kCal/mol$ and $\sigma_{Li} = 1.5050\dot{A}$ suggested by Bouzziz and Nasr (2007). For the Br- - Br-interaction, Sun et al. (2012) suggested that $\varepsilon_{Br} = 0.0899kCal/mol$ and $\sigma_{Br} = 4.6238\dot{A}$. The above values for the parameters are considered the most accepted and recent in the literature. The values of the potential parameters q, σ and ε are listed in Table 1.

Physical	00-	H+-H+	BrBr-	Li+- Li +
properties				
M (g/mol)	15.9994	1.008	79.9040	6.9400
q (ev)	-0.8476	+0.4238	-1.000	+1.000
Е	0.1533	0.0	0.0899	0.1649
(kCal/mol)				
σ(Å)	3.166	0.0	4.6238	1.5050

Table 1 MD Simulation Parameters.

The periodic boundary condition (PBC) is applied to all six boundary surfaces. The NVT ensemble was employed for the system absolute temperature that is maintained at different temperature levels by the Nose-Hoover thermostat developed by Nose (1984) and Hoover (1985). The truncated distance for short-range and long-range forces are taken as $12.5\dot{A}$ and $45\dot{A}$ respectively and the Ewald summation is employed for the Coulomb interaction calculation. The equations of motion are integrated using the Velocity-Verlet algorithm. The SHAKE method is applied for water molecules to maintain their bond lengths and angles. The number of molecules in each calculation set is summarized in Table 2.

The molecular dynamics simulation of water (Set I) and LiBr aqueous solution with three different concentrations (sets II, II and IV) were carried out. In the case of pure water (set I), the simulations were performed at three different temperatures, 293K, 373K and 473K. Simulations of the LiBr aqueous solutions (sets II, II and IV) were performed only at 473K.

For each system, the simulation was carried out with a time step of 1 fs and a total simulation time of 6 ns, comprised of 1 ns for equilibration and 5 ns for data analysis. The surface tension and distributions of ions were obtained by analyzing the trajectories recorded at every 500 fs.

Table 2 Number of Molecules Used in Each Calculation Set.

Set	H_20	Li+	Br-	Temperature (K)
Ι	400	0	0	293,373,473
II	400	16	16	473
III	400	40	40	473
IV	400	80	80	473

Further, MD simulation is used to simulate the absorption of water vapor into LiBr aqueous solutions. Fig. 4 provides a sketch of the simulation geometry.

Two boxes of the same dimensions $25\dot{A} \times 25\dot{A} \times 100\dot{A}$ are combined together at the beginning of the simulation to form the simulation system. The pervious equilibrium simulation result will be used as initial condition for this set of simulation.



Fig. 4 Schematic of combined simulation box for water vapor absorption into an aqueous electrolyte solution.

Initially, each box is under liquid-vapor equilibrium condition at different saturation vapor pressure. No external potential is applied and the pressure difference is the dominating driving force for the water vapor absorption into LiBr aqueous solution. The truncated distance for short-range and long-range forces are taken as $25\dot{A}$ and $40\dot{A}$ respectively. The equations of motion are integrated using the Velocity-Verlet algorithm. Similarly, the Ewald summation is also employed to calculate long range Coulomb interaction and SHAKE technique is utilized to hold the bond and angle between hydrogen and oxygen atoms in the water molecules.

The simulation conditions are all summarized in following Table 3. In the set of I-1,2,3, 4 and 5, MD of absorption of water vapor at 473K into LiBr aqueous solution with various concentrations and temperatures were carried out. The corresponding concentrations of LiBr aqueous solution are 0%, 49%, 59% and 71%, respectively. In order to get statistically sound result, four times of independent calculations are performed. In the set of II-1,2,3, and 4 and set of III-1,2,3, and 4, MD on water vapor molecules absorption into water at 383K and 71% LiBr aqueous solution at 473K were calculated.

All simulations are carried out on Sun Workstation (Ultra 27) with Intel Xeon Quad Core 3500 series and Nvidia GPU computing enabled. The molecular dynamics (MD) simulations of water vapor condensation on the surface of pure water or a LiBr aqueous solution with various solute concentrations were carried out by the open source Lammps code (Plimpton, 1995).

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	Evaporation side (A)		Absorption side (B)				
CASE	Number of molecules	Number of molecules		er of mole	Temperature[K]		
	H2O	[K]	H2O	Li+	Br-		
I-1	400	473	600	0	0	383	
I-2	400	473	400	60	60	400	
I-3	400	473	400	80	80	415	
I-4	400	473	400	120	120	437	
I-5	400	473	400	200	200	473	
II-1	400	423	600	0	0	383	
II-2	400	448	600	0	0	383	
II-3	400	473	600	0	0	383	
II-4	400	498	600	0	0	383	
III-1	400	423	400	200	200	473	
III-2	400	448	400	200	200	473	
III-3	400	473	400	200	200	473	
III-4	400	498	400	200	200	473	

3. RESULTS AND DISCUSSION

3.1 Effect of LiBr Concentration on the Rate of Vapor Condensation

Table 4 lists the MD simulation results that are specifically designed to evaluate the effects of various mass concentration levels of LiBr in the aqueous solution on the absorption of vapor molecules. For the initial pure water temperatures (evaporation side, Box A), they are all set at 473K for the five cases. For the absorption on pure water condition (Case I-1), its initial temperature is 383K for Box B. For the purpose of maintaining an equal condensation driving pressure difference for various LiBr concentrations, the initial temperatures of the LiBr solutions (condensing side, Box B) were assigned at 400K, 415K, 437K, and 473K for LiBr mass concentrations of 33% (Case I-2), 49% (Case I-3), 59% (Case I-4) and 71% (Case I-5), respectively. As the presence of LiBr lowers the vapor pressure, the solution temperature must be raised appropriately to maintain the same vapor pressure as that of the pure liquid water case.



between two boxes for the set of I-1, 2, 3, 4, and 5.

The MD simulation results are summarized in Table 4 and also plotted in Fig. 5 where $T_{A,i}$ and $T_{B,i}$, are initial box A temperature and initial Box B temperature, respectively. $N_{A\to B}$, $N_{B\to A}$, and N_{net} are total number of water molecules going from A to B, B to A and net from A to B during the first 60 ps, respectively. It is reminded that the noninteger numbers for the molecules transported are due to averaging over four independent simulations for a single case as mentioned previously. We need to mention that the uncertainties of the MD simulations associated with the numerical results presented in this paper are around ±15 to ±20% as the simulation was only based on a total of maximum 1400 molecules. In a similar MD simulation by Daiguji and Hihara (1999), the authors also estimated the errors are around ±15 to ±20%. Then, σ_C is defined as the condensation coefficient as follows:

$$\sigma_C = \frac{N_{net}}{N_{A \to B}} \tag{4}$$

To assess the effect of LiBr concentration based on the calculated condensation coefficients listed in Table 4, we found that the condensation coefficient is reduced by 10.9 %, 15.8%, 27.4% and 26.9% for LiBr mass concentration of 33%, 49%, 59% and 71%, respectively. It appears that the effect of LiBr concentration became saturated after 59%.

Table 4 MD Simulation Results on the LiBr Concentration Effects.

Case	LiBr Mass Concentration %	$T_{A,i}$ [K]	T _{B,i} [K]	$N_{A \rightarrow B}$	$N_{B \to A}$	N _{net}	σ _C
I-1	0%	473	383	50.5	5.5	45	0.891
I-2	33%	473	400	57	11.75	45.25	0.794
I-3	49%	473	415	61	15.25	45.75	0.750
I-4	59%	473	437	51.75	18.25	33.5	0.647
I-5	71%	473	473	46.25	16.15	30.10	0.651

3.2 Effect of Vapor Superheating on the Vapor Condensation

Next, we discuss the effect of superheating or temperature elevation of the water vapor over its saturation temperature on the condensation efficiency. To illustrate this effect, we chose a constant LiBr concentration of 71% and a constant condensing surface temperature of 473K while the vapor temperature is allowed to vary from 423K to 498K. As can be seen from Table 5, the vapor temperature has a relatively strong effect on the condensation coefficient. The condensation coefficient appears to saturate over 70% after the vapor temperature goes over 448K which corresponds to an increases about 11% when the vapor temperature is raised from 423K to 498K.

Table 5 MD Simulation Results on the Vapor Temperature Effects.

Case	LiBr Mass Concentration %	$T_{A,i}$ [K]	$T_{B,i}$ [K]	$N_{A \rightarrow B}$	$N_{B \to A}$	N _{net}	σ_{C}
II-1	71%	423	473	8	3	5	0.63
II-2	71%	448	473	21	5	16	0.76
II-3	71%	473	473	46	16	30	0.65
II-4	71%	498	473	75	19	56	0.75

The effect of vapor temperature is further analyzed by comparing with the vapor condensation on pure water results reported by Daiguji and Hihara (1999) with equal driving vapor pressure difference. Comparing between Tables 5 and 6 the only difference between Cases II and III, is the 71% LiBr concentration (Table 5) versus pure water (Table 6). The T_{B,i} temperature has to be lowed to 383K that is necessary to maintain the same vapor pressure difference for each subcase as explained before. Based on Table 6, it can be seen that the condensation coefficient starts out relatively high at 77% for a vapor temperature of 423K and then it apparently saturates around 90% as water vapor reaches 448K. The Case of T_{A,i} at 498K actually has a lower value of σ_c which is mainly due the uncertainty of the MD simulations mentioned above.

The comparison of condensation coefficients between Cases II and III is given in Table 7. It is clear that the effect of LiBr concentration on reducing the condensation coefficient is becoming less significant as the vapor temperature is increased. In summary, Table 7 provides a list for all the condensation coefficients for Cases II and III. The condensation coefficient increases with vapor temperature for both cases. However, as long as the LiBr concentration is kept constant at 71%, the ratio of respective condensation coefficients is relatively constant ~ 85% irrelevant to vapor temperature variation.

Table 6	MD	Simulation	Results or	n the	Vapor	Temperature	Effects	foi
Condens	ation	on Pure W	ater from I	Daigu	iji and l	Hihara (1999)).	

Case	LiBr Mass Concentration %	$T_{A,i}$ [K]	$T_{B,i}$ [K]	$N_{A \rightarrow B}$	$N_{B \to A}$	N _{net}	σ_{C}
III-1	0%	423	383	13	3	10	0.77
III-2	0%	448	383	29	3	26	0.90
III-3	0%	473	383	48	6	42	0.88
III-4	0%	498	383	109	15	94	0.86

Table 7. Comparison between Case II and Case III.

Case	Subcase 1, σ_c	Subcase 2, σ_c	Subcase 3, σ_c	Subcase 4, σ_c
II	0.63	0.76	0.65	0.75
III	0.77	0.90	0.88	0.86
II/III	82%	84%	74%	87%
0/0				

3.3 The Effect of LiBr on the Average Condensation Coefficient

As presented in the previous section, the temperature of the vapor phase holds a significant effect on the vapor condensation effectiveness. From a theoretical point of view, Tsuruta and Nagayama (2004) and others (Tsuruta et al., 1995; Tsuruta et al., 1999) have proposed and later verified that the condensation phenomenon is affected by the energy levels of the vapor phase. Specifically this effect is mainly dominated by the normal component of the translational kinetic energy of the molecule that is perpendicular to the condensing surface, Ez, that is striking the interface. Furthermore, Tsuruta and Nagayama (2004) proposed the equation below to correlate the condensation coefficient with Ez at the vapor-liquid interface temperature, T_s

$$\sigma_{C} = \alpha (1 - \beta e^{-\frac{E_{Z}}{K_{B} T_{S}}})$$

$$E_{Z} = \frac{1}{m} W_{Z}^{2}$$
(5)
(6)

Where α and β are a set of constants for a given T_s. K_B is the Boltzmann constant and m and V_Z are the mass of the vapor molecule and its normal component velocity, respectively.

Once the functional dependence of the condensation coefficient on the Ez is determined, we can evaluate the average condensation coefficient if the energy distribution form is known. Tsuruta and Nagayama (2004) suggested the Maxwellian distribution as it is generally valid for most gases. The ensemble average condensation coefficient, $\overline{\sigma_c}$ is then calculated based on the following equation using the Maxwellian energy distribution :

$$\bar{\sigma}_{C} = \frac{1}{(K_{B}T_{s}/2\pi m)^{1/2}} \int_{0}^{\infty} \sigma_{C} V_{z} \left(\frac{m}{2\pi K_{B}T_{s}}\right)^{1/2} \exp(-\frac{E_{z}}{K_{B}T_{s}}) dV_{z} = \alpha (1 - \beta / 2)$$
(7)

It is noted the normal component of the translational energy is assumed to be one third of the total energy due to the vapor temperature,

$$E_Z = \frac{1}{2} K_B T_v \tag{8}$$

where T_v is the vapor absolute temperature.

Based on the above, we have used the results of Case II and Case III given in Tables 5 and 6, respectively, to find the average condensation coefficients for pure water and 71% LiBr solution. Using the data from Tables 5 and 6 to perform curve fitting for Equation (5). The curve fitting results are shown in Fig. 6. that determine the constants, α and β . The

constants, α and β , together with the average condensation coefficient are given in Table 8 for pure water and 71% LiBr solution.

Based on Table 8, first we note that the average condensation coefficient for water condensing on pure water at 383K is 0.745 that is in good agreement with those given in the open literature (Tsuruta and Nagayama, 2004; Bonacci et al., 1976). The average condensation coefficient is reduced by 43.76% at 0.77 for the 71% LiBr aqueous solution.

Table 8 Average condensation coefficient

Case	α	β	$ar{\sigma}_{_C}$
II. Pure water	1.6551	0.8860	0.9219
III. 71% LiBr solution	1.5083	0.9790	0.7700



Fig. 6 Condensation coefficient vs. E_Z/K_B for pure water and 71% LiBr solution, (a) expanded domain and (b) focused domain.

4. COMPARISON OF NANOSCALE VERSUS MACROSCALE VAPOR ABSORPTION

4.1 Absorption Mass Fluxes in Macro Systems

In most of the existing engineering systems, the macroscale absorbers are generally designed in the forms of either a vertical falling film type (Fig. 7a) or a horizontal tube array type (Fig. 7b). Both numerical and experimental results for the macroscale vapor absorption rates of vertical and horizontal tube-type absorbers (Fig. 7) are summarized in Tables 9 and 10 and discussed as follows. Ideally, it would be desirable to compare the absorption rates under the same driving force of vapor pressure difference between the LiBr solution and the vapor chamber. However, due to different system configurations and operating conditions in those experiments and numerical simulations, it becomes relatively meaningless to compare the absorption rate per vapor pressure difference.



Fig. 7 Vertical and horizontal tube absorbers

In Tables 9 and 10, the vapor absorption mass fluxes (mass flow rate per unit area) together with the most important parameters are listed to give the readers necessary information to compare the results. Table 9 summarizes the vapor mass absorption fluxes for the vertical falling film type, such as vertical wall and tube absorbers. The range of absorption rates is from 0.00125 to 0.00322 kg/m²/s. The mass fluxes of vertical tubes tend to be twice as large as those of the vertical plate. This is not surprising as the length scale of a tube type is usually smaller than that of a plate type even though both are in the category of macroscale.

Table 9 Mass absorption rates in vertical wall and tube absorbers

Referenc	Absorption	Inlet	Solution	Lengt	Diamete	exp.
е	rate	Concentra	Rate	h [m]	r [m]	or
	$[kg/m^2/s]$	tion	[Kg/m/s]			пит
Miller &	0.00322	62%	0.02 kg/s	1.524	0.01905	exp.
Keyhani						
(2001)						
Tsai &	0.003	62%	-	1.5	0.01905	exp.
Blanco						-
(1998)						
Matsuda	0.0027	55%	0.1	-	vertical	exp.
et al.					plate	-
(1994)					-	
Bo et al.	0.00125	60%	0.014	1.0	vertical	num
(2010)					plate	
Patnaik et	0.0017	60%	0.02	2.4	vertical	num
al. (1993)					plate	

Table 10 Mass absorption rates in horizontal tubular absorbers

Reference	Absorption rate [kg/m ² /s]	Inlet Concentration [%]	Solution Rate [kg/m/s]	Diameter [m]	Exp or num
Choudhury et al. (1993)	0.0026	60.0	0.01	0.021	num
Jeong & Garimella (2002)	0.0023	60.1	0.058	0.016	num
Sultana et al. (2007)	0.0022	60.4	0.044	0.022	exp
Islam et al. (2003)	0.0026	60.4	0.044	0.022	num
Islam (2008)	0.0022	60.4	0.05	0.019	exp
Yoon et al. (2008)	0.0025	61.0	0.030	0.01588	exp

Table 10 shows mass absorption fluxes for horizontal tube-type absorbers and this type is the most common geometry used in absorption chillers. It can be seen that there is a better agreement of absorption fluxes in this table ($0.0022-0.0026 \text{ kg/m}^2/\text{s}$). It is noted that even with relatively large variations in the geometry and system conditions, among the vapor absorption fluxes, there is generally a good agreement. In summary, the average absorption flux for common horizontal tube absorbers is around

0.0024 Kg/m²/s, while this value for vertical tube absorbers is around 0.003 Kg/m²/s. Also we can conclude that for the macroscale vapor absorption flux, its order of magnitude is ~ $O(10^{-3})$ kg/m²s.

4.2 Comparison between Nanoscale and Macroscale Absorption Fluxes

In this section, we will evaluate the nanoscale vapor condensation mass fluxes at the vapor-liquid interface and compare with those of the macroscale. The results presented in Table 4 on the net total number of vapor molecules going from Box A to Box B in 60 ps are converted to mass fluxes as we know the cross sectional area between the two boxes is 2.5 nm x 2.5 nm. The converted interfacial condensation mass fluxes are provided in Table 11.

Table 11 Nanosca	le condensation	mass fluxes
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Case	LiBr Mass Concentration %	$T_{A,i}$ [K]	$T_{B,i}$ [K]	$N_{A \rightarrow B}$	$N_{B \to A}$	N _{net}	ṁ [kg/m²s]
I-3	49%	473	415	61	15.25	45.75	3,651
I-4	59%	473	437	51.75	18.25	33.5	2,673
I-5	71%	473	473	46.2	16.15	30.05	2,398
I-1	0%	473	383	50.5	5.5	45	3,591

We can see that the mass fluxes are in the range of 2,398 to 3,651 kg/m²s or these fluxes are on the order of ~ $O(10^3)$ kg/m²s. Compared with the macroscale absorption mass fluxes of ~ $O(10^{-3})$ kg/m²s, the nanoscale mass fluxes are six orders of magnitude larger. First we need to point out that the nanoscale fluxes at the level of 10^3 kg/m^2 s have been reported by several previous publications. For example, Tsuruta and Nagayama (2004) calculated the mass fluxes on the order of 2,000 kg/m²s for water vapor condensation on pure liquid water surface in a nanoscale system. For a nanoscale evaporating argon liquid film, Maroo and Chung (2010) predicted the mass fluxes around 1,000 kg/m²s. The physical explanation is that in these nanoscale phase change processes, the scale is around 10 nm (10⁻⁸ m). While the macroscale systems mentioned in Section 4.1, the transport scale is on the order of 10⁻² m, that basically verifies the six orders of magnitude difference in the transport rates as in theory the heat and mass transport is inversely proportional to the length scale.

5. CONCLUSION

In this paper, we focus on the physical understanding and the evaluation of the condensation efficiency for the nanoscale absorption of water vapor at the vapor-liquid interface through a condensation process. A detailed analysis based on molecular dynamics simulation results has been performed to evaluate the effects of LiBr concentration in the aqueous solution on the water vapor condensation rate and to compare the nanoscale condensation rates with those of macroscale.

To assess the effect of LiBr concentration based on the calculated condensation coefficients, we found that the condensation rate is reduced by 10.9 %, 15.8%, 27.4% and 26.9% for LiBr mass concentration of 33%, 49%, 59% and 71%, respectively.

Under a constant LiBr concentration of 71% and a constant condensing surface temperature of 473K, the condensation coefficient appears to saturate over 70% after the vapor temperature goes over 448K which corresponds to an increase of about 11% when the vapor temperature is raised from 423K to 498K.

It is clear that the effect of LiBr concentration on reducing the condensation coefficient is becoming less significant as the vapor temperature is increased. The condensation coefficient increases with vapor temperature for both pure water and LiBr aqueous solution cases. However, as long as the LiBr concentration is kept constant at 71%, the

ratio of respective condensation coefficients is relatively constant ~ 85% irrelevant to vapor temperature variation.

A model is also developed to predict the average condensation coefficient for the complete vapor energy spectrum. For condensation on the 71% LiBr aqueous solution, the average condensation coefficient is reduced by 43.76% at 77% as compared to that for the condensation on pure liquid water.

For the nanoscale condensation mass fluxes, they are on the order of $O(10^3)$ kg/m²s while those of the macroscale are of the order $O(10^{-3})$ kg/m²s, that is six orders of magnitude smaller and that also indicates the strong effect of the length scale.

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NOMENCLATURE

II	Field notential (V)
U	i leia potentiai ()

- q particle charge (C)
- r distance (Å)
- M molar mass (kg/kmol)
- T temperature (K)
- N Number of molecules
- *E* translational kinetic energy of the molecule (J)
- \dot{m} mass flux (kg/m²s)

Greek Symbols

σ	the distance at which the intermolecular potential between
	the two particles is zero (\dot{A})
3	Well depth (V)

Subscripts

С	LiBr concertation
Net	Net amount of molecules moving from Box A to Box B
Α	Box A
В	Box B
S	Vapor-liquid interface
v	Vapor

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